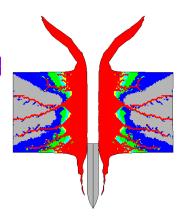
Modal Testing as an Aid in sessing Penetration Mechanic Patrick L. Walter

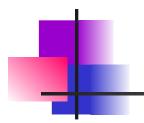
Endevco, San Juan Capistrano, CA



TCU, Fort Worth, T>



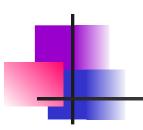




Goals

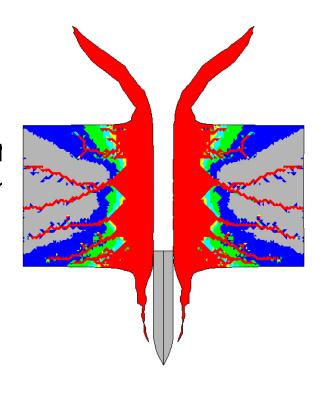
- Describe a gun-launched earth penetrator test
 - provide test details
 - review and analyze test results
 - draw conclusions concerning analytical/experimental process
- Support program theme
 - measurement system design
 - transducers, system checks, model verification, modal analysis, data filtering, data sampling,



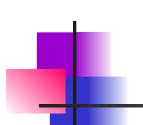


Earth Penetrator Applications

- Deliver ordnance device
- Exploration of geological layer
- Measurement of sea ice thickr
- Insitu chemistry
- etc.







Specific Penetrator to be Field Tested

- length: 61.25"
- diameter: 6.125"
- wall thickness: 1.062"
- weight: 336 pounds
- c.g.: 28.05" from nose
- on board data recording system (accelerometer triggered)
 - resolution: *6 bits (1 part in 63)
 - Nyquist frequency: 11,300 Hz
 - two data channels
 - anti-alias filters designed

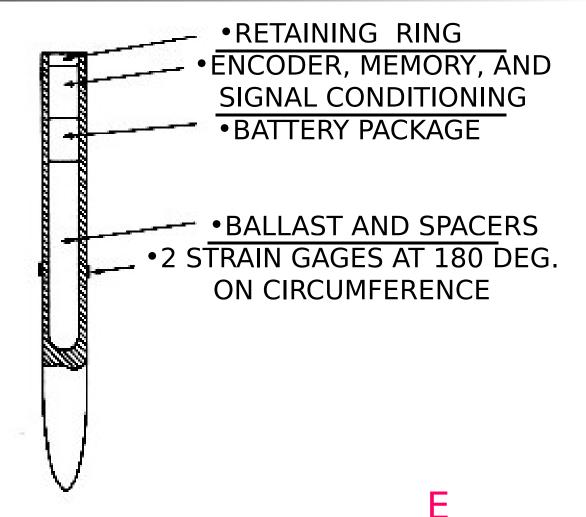


Specific Penetrator (con't)

- two data channels (con't)
 - Micro-Measurements WA 06-250BK-10C strain gages (constantan material, 1/4" gage length, temperature compensated for steel, fully encapsulated, 1000Ω resistance)
 - 180 degrees on circumference, 30" from nose
 - measure compression and bending strain calibrated +/- 6,000με (corresponds to yield of penetrator steel case [D6 A-C normalized and

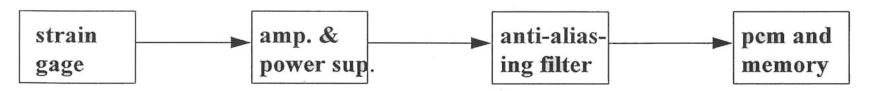


Specific Penetrator (con't)





Specific Penetrator (con't)



MM WA 06-250BK-10C +/- 6,000 microstrain full scale

- 6dB at 4,200 Hz 24 dB/octave

~211 microstrain/word

constantan 1,000 ohm T. C. 6 ppm/deg F

.250" grid length encapsulated in glass fiber reinforced epoxy phenolic resin ~48,000 bits

~6 bits/word

~179 msec. window dT/sample for 2

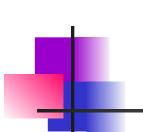
channel system =

.00004417 sec. =

11,300 Hz Nyquist

frequency





Experimental modal Analysis (review)

- Experimental modal analysis enables extraction of:
 - shape,
 - natural frequency, and
 - damping

for each vibratory mode of a structi

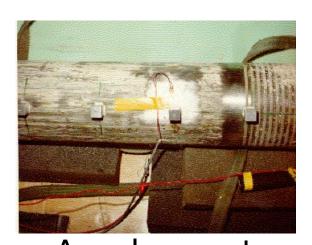


- NASTRAN
- ANSYS
- ALGOR





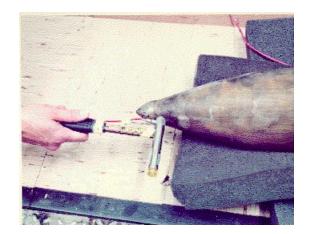
Penetrator Experimental Modal Analysis Results Prior to Field Test



Acceleromet er Mounting cations



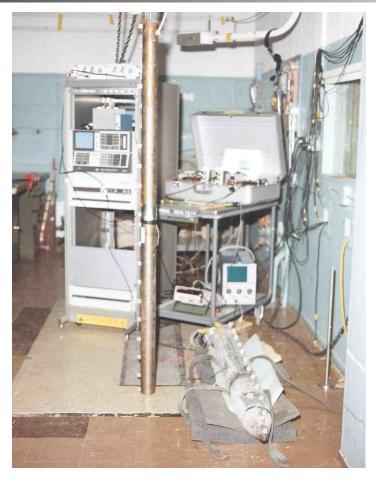
Penetrator tested with Hammer Input Free-Free Boundaries



Instrumented Sandia



Penetrator Experimental Modal Analysis Results Prior to Field Test





Sandia

deterministic structure adds credibility





Fourth Bending Mode - 2,713 HЪ

Penetrator Natural Frequencies

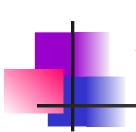
<u>Bending</u>	<u>Axial</u>
392	1,712
976	3,845

1,764

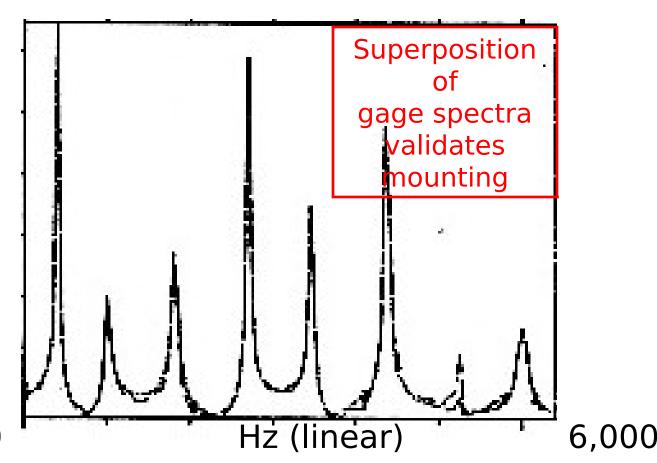
4,368

2,713 Agree with Analytical Model 3,464





Strain gage Mounting Verification By Modal Test

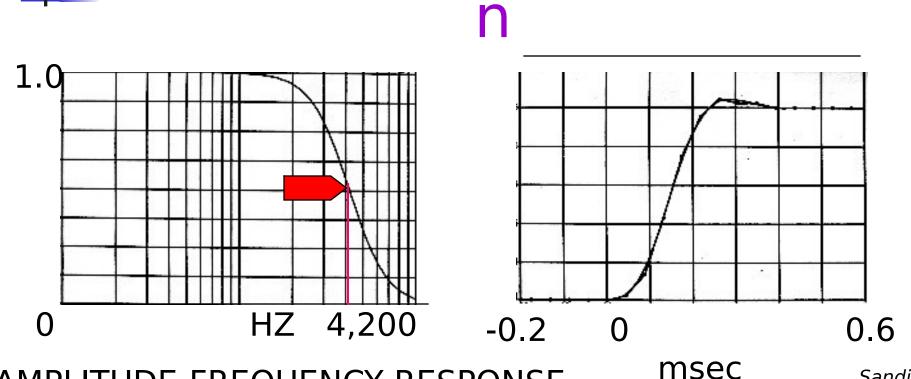




Sandia

E

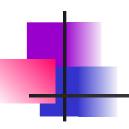
Field Data Recording System Characterization/Verificatio



AMPLITUDE-FREQUENCY RESPONSE

Sandia

(2 amplitude levels > linearity verified) IT STEP RESPONSE (bit resolution $213\mu\epsilon$)



Penetrator Preparation



strain gage mounted before encapsulation

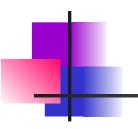




Penetrator Preparation





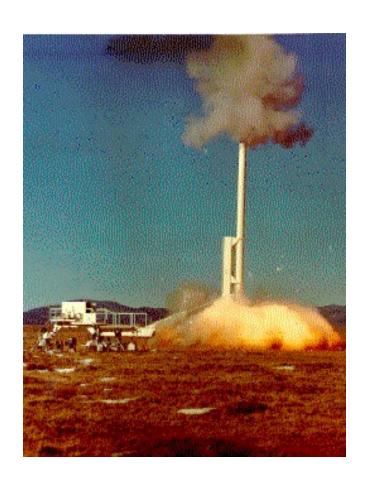


Penetrator Preparation







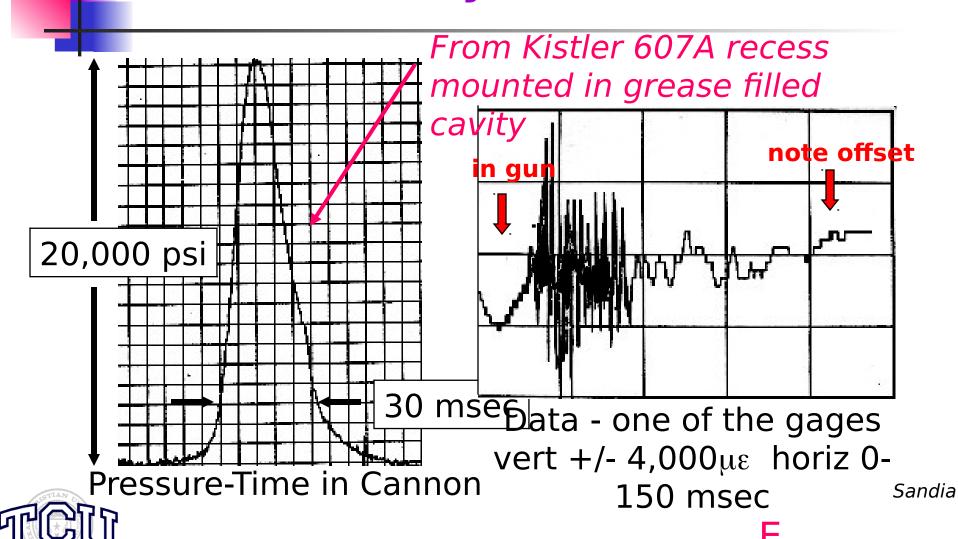


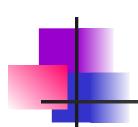
Davis Gun

- recoilless Cannon
- 2 Deg to Vertical
- pressure data TM from barrel transducer (Kistler 607A in grease filled cavity)
- 93' dry lake bed target penetration



Data Analysis/Validation



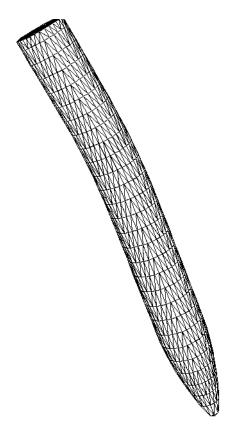


Calculation Consideration:

- from peak pressure in gun:
 - sabot area, yields peak force
- total mass penetrator
 - yields total peak acceleration
- penetrator characteristics:
 - cross section area
 - modulus of elasticity
 - mass in front of strain gage
 - enables calculation of 2,040με <u>VS</u>
 2,000 2,213με (within bit resolution)

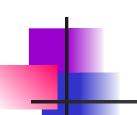


Data offset explanation:



Miners report that penetrator springs back when freed





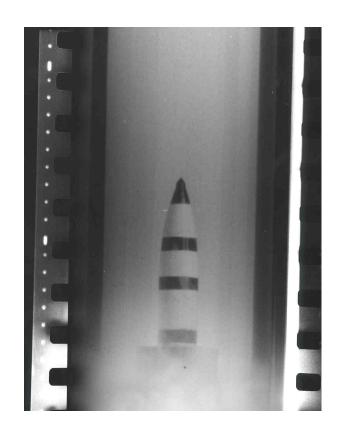
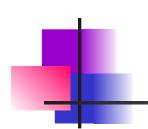
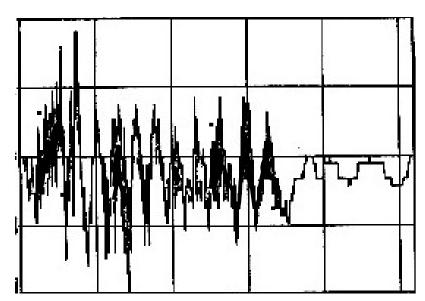


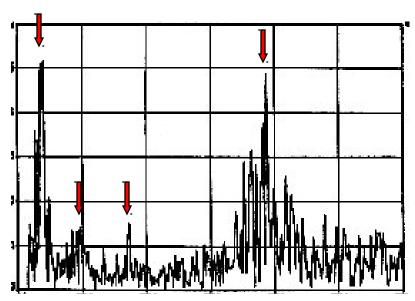


image motion also helps in diagnostics



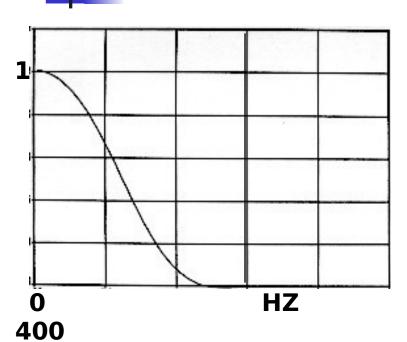


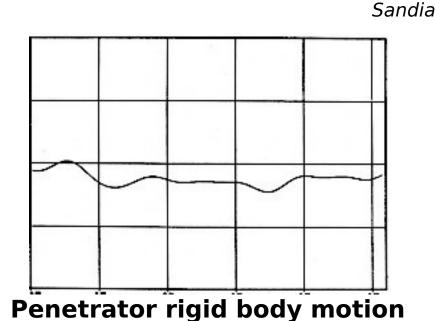
Data Time Expanded - one of the gag vert +/- 4,000με horiz 0-73 msec



Fourier transform of same horiz 0-6,000 Hz peaks at ~ 392, 976, 1712, & 3845 Hz



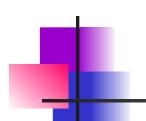




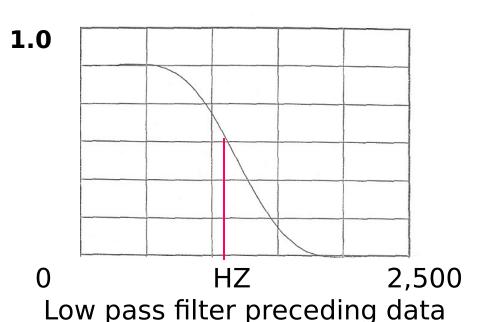
vert \pm 4,000 $\mu\epsilon$ horiz 0-73 msec

Low pass filter preceding

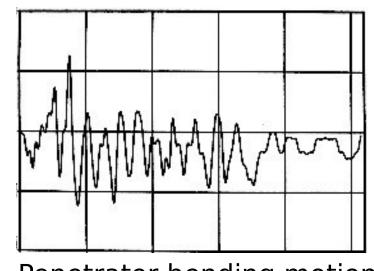
dateulate constant deceleration (assumption) required on penetrator to stop in 93'. Combine with E, cross section area & mass in front of calculate -300με. Note: within bit levels







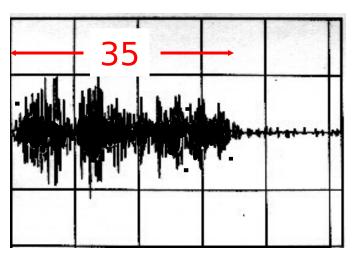
(1,100 Hz - 3dB point)



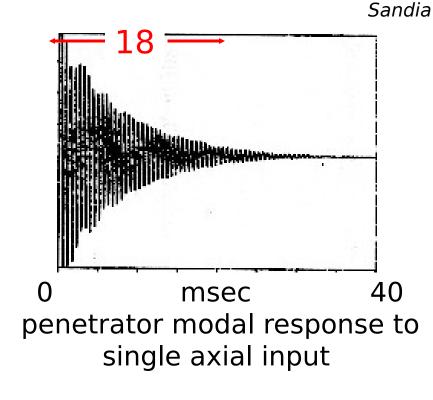
Penetrator bending motion vert +/- 4,000με horiz 0-73 msec

Zero phase shift filter enables waveform subtraction (see next)





penetrator high frequency
Axial motion
(subtracted from original data)
vert +/- 4,000με horiz 0-73

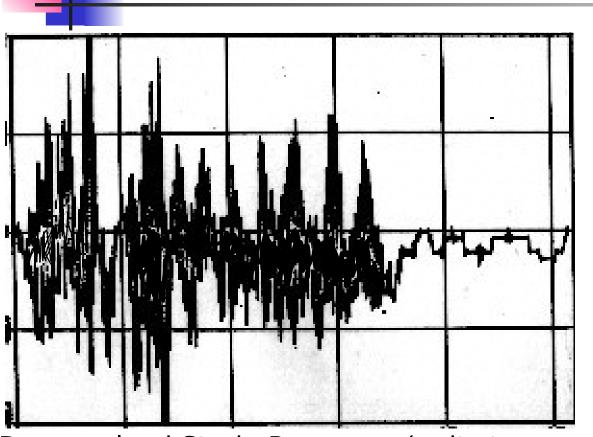


Conclusion: High frequency axial loading is occurring



many body lengths of penetration.

Data Analysis



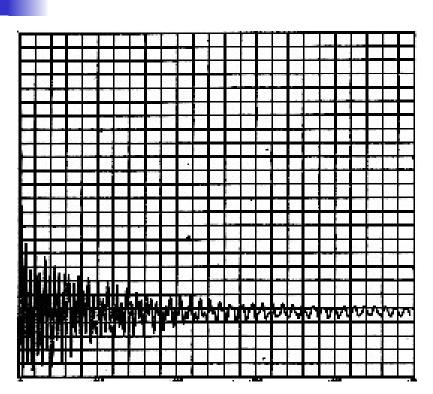
- increased high frequency
- increased amplitud

Deconvolved Strain Response (unit step used)

vertical $\pm /-6,000 \mu \epsilon$ horiz 0-73 msec



Data Analysis/Pretest Predictions

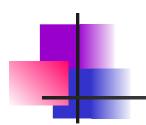


- note significant difference
- large bending and axial strains that occurred duri test differed greatly from analytical predictions!

Sandia

Analytically predicted test results vertical +/- 400με horiz 0 - 50 msec





Conclusions

- Pretest, experimental modal analysis results agreed with analytical structural model.
- Strain gages were verified to be properly mounted.
- Data recording system was dynamically characterized and verified to
 - be linear.
- Independent post-test calculations based on pressure-time in gun
 - and depth of penetration correlated with measured strain
- ♥Aeanalytical loads applied to the penetrator in the modeling Brotiggificant differentesimproversing tradtreapseuribetween
- Tepeniged on more representative models for the